

Stress and Strain Measures

Goals - Stress & Strain Measures

- Definition of a nonlinear elastic problem
- Understand the deformation gradient?
- What are Lagrangian and Eulerian strains?
- What is polar decomposition and how to do it?
- How to express the deformation of an area and volume
- What are Piola-Kirchhoff and Cauchy stresses?

What Is a Nonlinear Elastic Problem?

- **Elastic** (same for linear and nonlinear problems)
 - Stress-strain relation is elastic
 - Deformation disappears when the applied load is removed
 - Deformation is history-independent
 - Potential energy exists (function of deformation)

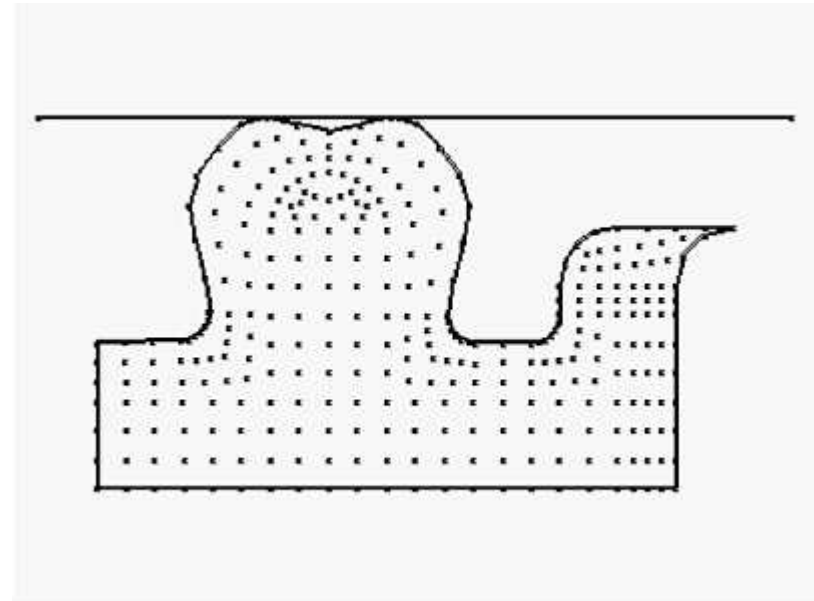
- **Nonlinear**
 - Stress-strain relation is nonlinear
 - Deformation is large

- **Lagrangian or Material Stress/Strain:**

when the reference frame is
undeformed configuration

- **Eulerian or Spatial Stress/Strain:**

when the reference frame is deformed configuration

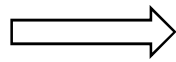


Deformation and Mapping

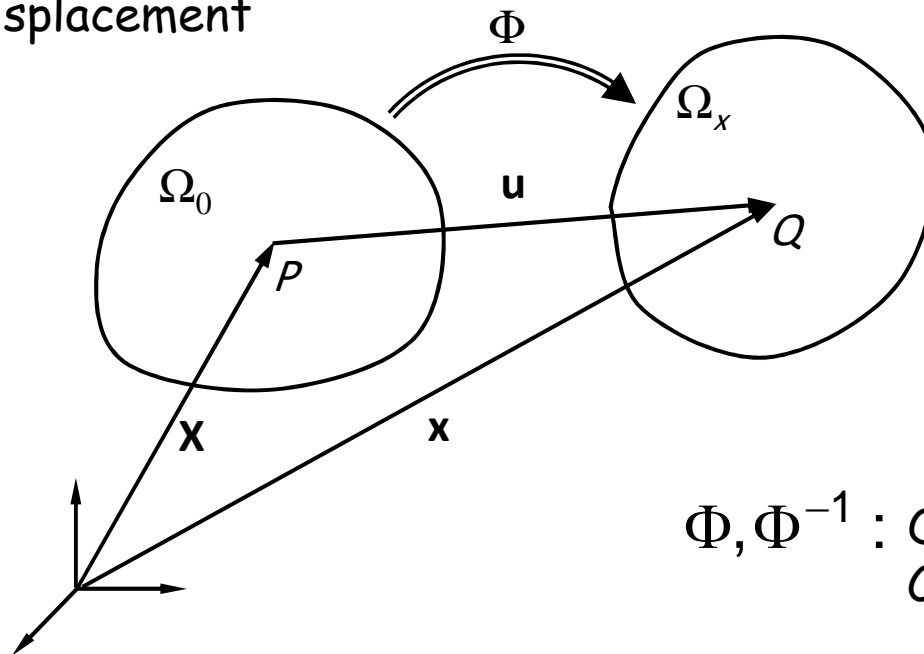
- Initial domain Ω_0 is deformed to Ω_x
 - We can think of this as a **mapping** from Ω_0 to Ω_x
- \mathbf{X} : material point in Ω_0 \mathbf{x} : material point in Ω_x
- Material point P in Ω_0 is deformed to Q in Ω_x

$$\mathbf{x} = \mathbf{X} + \mathbf{u}$$

↑
displacement



$$\mathbf{x} = \Phi(\mathbf{X}, t) = \mathbf{X} + \mathbf{u}(\mathbf{X}, t)$$

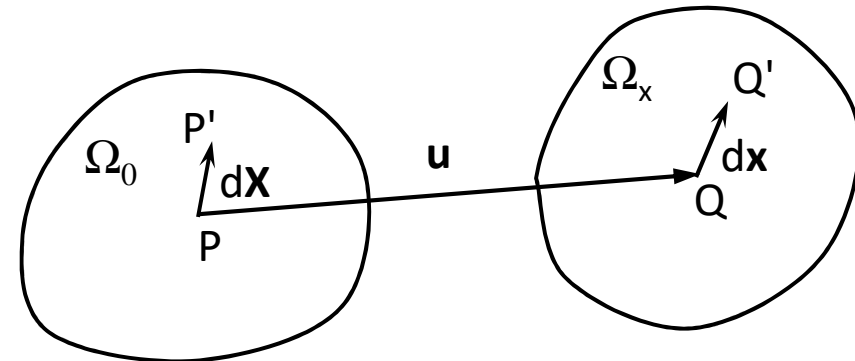


Φ, Φ^{-1} : One-to-one mapping
Continuously differentiable

Deformation Gradient

- Infinitesimal length $d\mathbf{X}$ in Ω_0 deforms to $d\mathbf{x}$ in Ω_x
- Remember that the mapping is **continuously differentiable**

$$d\mathbf{x} = \frac{\partial \mathbf{x}}{\partial \mathbf{X}} d\mathbf{X} \Rightarrow d\mathbf{x} = \mathbf{F} d\mathbf{X}$$



- **Deformation gradient:**

$$F_{ij} = \frac{\partial x_i}{\partial X_j}$$

$$\mathbf{F} = \mathbf{1} + \frac{\partial \mathbf{u}}{\partial \mathbf{X}} = \mathbf{1} + \nabla_0 \mathbf{u}$$

$$\mathbf{1} = [\delta_{ij}],$$

$$\nabla_0 = \frac{\partial}{\partial \mathbf{X}}, \quad \nabla_x = \frac{\partial}{\partial \mathbf{x}}$$

- gradient of mapping Φ

- **Second-order tensor, Depend on both Ω_0 and Ω_x**

- Due to one-to-one mapping: **$\det \mathbf{F} \equiv J > 0$.**

$$d\mathbf{X} = \mathbf{F}^{-1} d\mathbf{x}$$

- **\mathbf{F} includes both deformation and rigid-body rotation**

Example - Uniform Extension

- Uniform extension of a cube in all three directions

$$x_1 = \lambda_1 X_1, \quad x_2 = \lambda_2 X_2, \quad x_3 = \lambda_3 X_3$$

- **Continuity requirement:** $\lambda_i > 0$

- Deformation gradient:

$$\mathbf{F} = \begin{bmatrix} \lambda_1 & 0 & 0 \\ 0 & \lambda_2 & 0 \\ 0 & 0 & \lambda_3 \end{bmatrix}$$

- $\lambda_1 = \lambda_2 = \lambda_3$: uniform expansion (**dilatation**) or contraction

- Volume change

- Initial volume: $dV_0 = dX_1 dX_2 dX_3$

- Deformed volume:

$$dV_x = dx_1 dx_2 dx_3 = \lambda_1 \lambda_2 \lambda_3 dX_1 dX_2 dX_3 = \lambda_1 \lambda_2 \lambda_3 dV_0$$

Green-Lagrange Strain

- Why different strains?

- Length change: $\|dx\|^2 - \|dX\|^2 = dx^T dx - dX^T dX$
 $= dX^T F^T F dX - dX^T dX$
 $= dX^T \underbrace{(F^T F - 1)}_{\text{Ratio of length change}} dX$

- **Right Cauchy-Green Deformation Tensor**

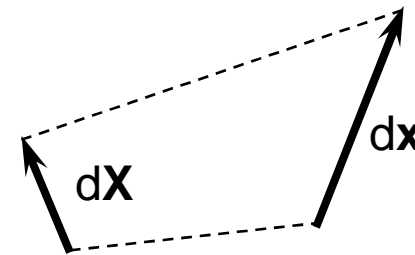
$$C = F^T F$$

- **Green-Lagrange Strain Tensor**

$$E = \frac{1}{2}(C - 1)$$



To match with infinitesimal strain



The effect of rotation is eliminated

Green-Lagrange Strain cont.

- Properties:

- \mathbf{E} is **symmetric**: $\mathbf{E}^T = \mathbf{E}$

- No deformation: $\mathbf{F} = \mathbf{1}$, $\mathbf{E} = \mathbf{0}$

$$\varepsilon_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial X_j} + \frac{\partial u_j}{\partial X_i} \right)$$

$$\mathbf{E} = \frac{1}{2} \left(\frac{\partial \mathbf{u}}{\partial \mathbf{X}} + \frac{\partial \mathbf{u}^T}{\partial \mathbf{X}} + \frac{\partial \mathbf{u}^T}{\partial \mathbf{X}} \frac{\partial \mathbf{u}}{\partial \mathbf{X}} \right)$$

$$= \frac{1}{2} \left(\nabla_0 \mathbf{u} + \nabla_0 \mathbf{u}^T + \nabla_0 \mathbf{u}^T \nabla_0 \mathbf{u} \right)$$

Displacement gradient

Higher-order term

- When $|\nabla_0 \mathbf{u}| \ll 1$, $\mathbf{E} \approx \frac{1}{2} (\nabla_0 \mathbf{u} + \nabla_0 \mathbf{u}^T) = \boldsymbol{\varepsilon}$

- $\mathbf{E} = \mathbf{0}$ for a rigid-body motion, but $\boldsymbol{\varepsilon} \neq \mathbf{0}$

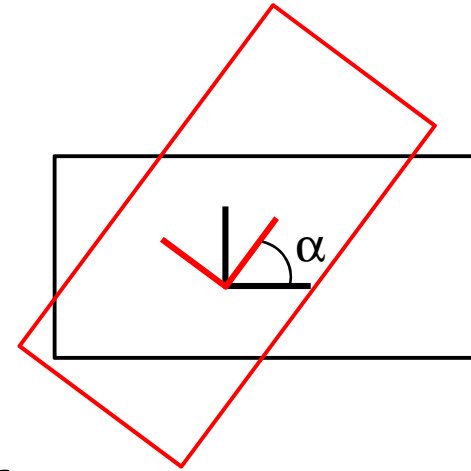
Example - Rigid-Body Rotation

- Rigid-body rotation

$$x_1 = X_1 \cos \alpha - X_2 \sin \alpha$$

$$x_2 = X_1 \sin \alpha + X_2 \cos \alpha$$

$$x_3 = X_3$$



- Approach 1: using deformation gradient

$$\mathbf{F} = \begin{bmatrix} \cos \alpha & -\sin \alpha & 0 \\ \sin \alpha & \cos \alpha & 0 \\ 0 & 0 & 1 \end{bmatrix} \quad \mathbf{F}^T \mathbf{F} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}$$

$$\mathbf{E} = \frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{1}) = \mathbf{0}$$

Green-Lagrange strain removes rigid-body rotation from deformation

Example - Rigid-Body Rotation cont.

- Approach 2: using displacement gradient

$$u_1 = x_1 - X_1 = X_1(\cos \alpha - 1) - X_2 \sin \alpha$$

$$u_2 = x_2 - X_2 = X_1 \sin \alpha + X_2(\cos \alpha - 1)$$

$$u_3 = x_3 - X_3 = 0$$

$$\nabla_0 \mathbf{u} = \begin{bmatrix} \cos \alpha - 1 & -\sin \alpha & 0 \\ \sin \alpha & \cos \alpha - 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$\nabla_0 \mathbf{u}^T \nabla_0 \mathbf{u} = \begin{bmatrix} 2(1 - \cos \alpha) & 0 & 0 \\ 0 & 2(1 - \cos \alpha) & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

$$\mathbf{E} = \frac{1}{2}(\nabla_0 \mathbf{u} + \nabla_0 \mathbf{u}^T + \nabla_0 \mathbf{u}^T \nabla_0 \mathbf{u}) = \mathbf{0}$$

Example - Rigid-Body Rotation cont.

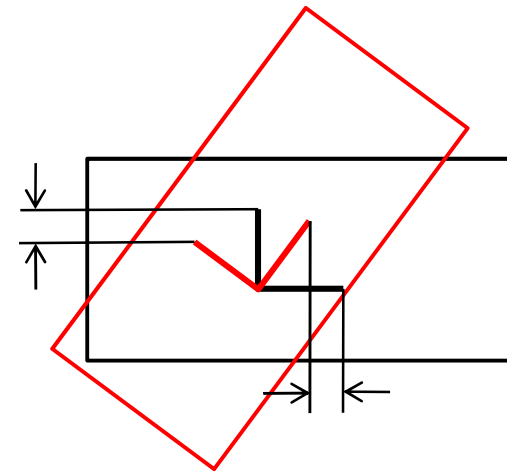
- What happens to engineering strain?

$$u_1 = x_1 - X_1 = X_1(\cos \alpha - 1) - X_2 \sin \alpha$$

$$u_2 = x_2 - X_2 = X_1 \sin \alpha + X_2(\cos \alpha - 1)$$

$$u_3 = x_3 - X_3 = 0$$

$$v = \begin{bmatrix} \cos \alpha - 1 & 0 & 0 \\ 0 & \cos \alpha - 1 & 0 \\ 0 & 0 & 0 \end{bmatrix}$$



Engineering strain is unable to take care of rigid-body rotation

Eulerian (Almansi) Strain Tensor

- Length change: $\|dx\|^2 - \|dX\|^2 = dx^T dx - dX^T dX$
 $= dx^T dx - dx^T F^{-T} F^{-1} dx$
 $= dx^T (1 - F^{-T} F^{-1}) dx$
 $= dx^T (1 - b^{-1}) dx$

- Left Cauchy-Green Deformation Tensor

$$\mathbf{b} = \mathbf{F}\mathbf{F}^T$$

\mathbf{b}^{-1} : Finger tensor

- Eulerian (Almansi) Strain Tensor

$$\mathbf{e} = \frac{1}{2}(\mathbf{1} - \mathbf{b}^{-1})$$

Reference is deformed (current) configuration

Eulerian Strain Tensor cont.

- Properties

- Symmetric

- Approach engineering strain when $\frac{\partial \mathbf{u}}{\partial \mathbf{x}} \ll 1$

- In terms of displacement gradient

$$\mathbf{e} = \frac{1}{2} \left(\frac{\partial \mathbf{u}}{\partial \mathbf{x}} + \frac{\partial \mathbf{u}^T}{\partial \mathbf{x}} - \frac{\partial \mathbf{u}^T}{\partial \mathbf{x}} \frac{\partial \mathbf{u}}{\partial \mathbf{x}} \right)$$
$$= \frac{1}{2} \left(\nabla_x \mathbf{u} + \nabla_x \mathbf{u}^T - \nabla_x \mathbf{u}^T \nabla_x \mathbf{u} \right)$$

$$\nabla_x = \frac{\partial}{\partial \mathbf{x}}$$

Spatial gradient

- Relation between \mathbf{E} and \mathbf{e}

$$\mathbf{E} = \mathbf{F}^T \mathbf{e} \mathbf{F}$$

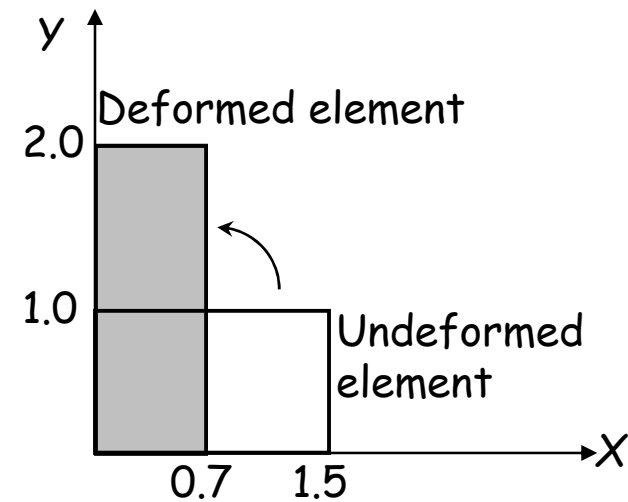
Example - Lagrangian Strain

- Calculate \mathbf{F} and \mathbf{E} for deformation in the figure
- Mapping relation in Ω_0

$$\begin{cases} X = \frac{3}{4}(s + 1) \\ Y = \frac{1}{2}(t + 1) \end{cases}$$

- Mapping relation in Ω_x

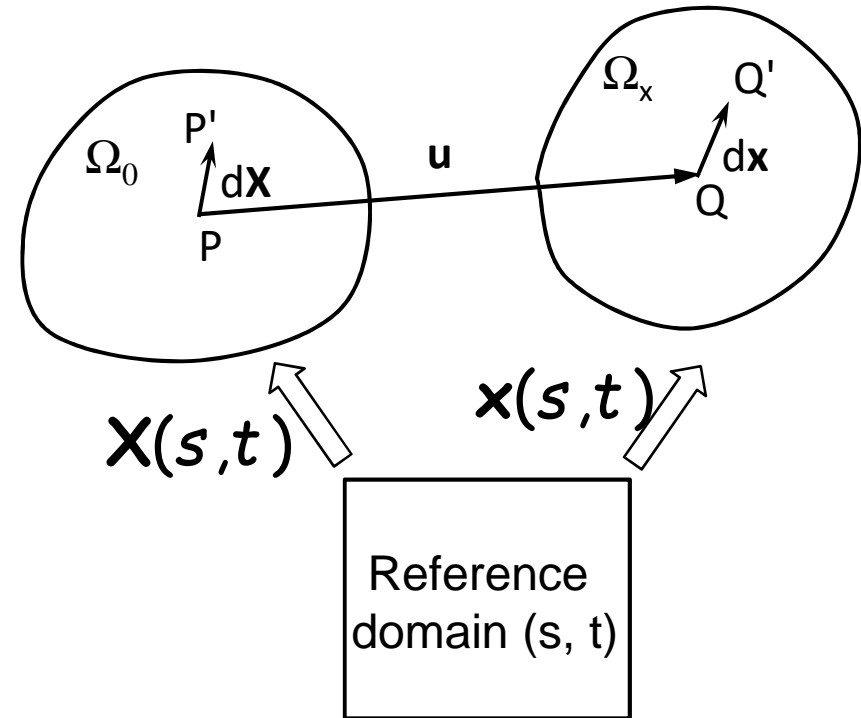
$$\begin{cases} x(s, t) = 0.35(1 - t) \\ y(s, t) = s + 1 \end{cases}$$



Example - Lagrangian Strain cont.

- Deformation gradient

$$\begin{aligned} \mathbf{F} &= \frac{\partial \mathbf{x}}{\partial \mathbf{X}} = \frac{\partial \mathbf{x}}{\partial \mathbf{s}} \frac{\partial \mathbf{s}}{\partial \mathbf{X}} \\ &= \begin{bmatrix} 0 & -0.35 \\ 1 & 0 \end{bmatrix} \begin{bmatrix} 4/3 & 0 \\ 0 & 2 \end{bmatrix} \\ &= \begin{bmatrix} 0 & -0.7 \\ 4/3 & 0 \end{bmatrix} \end{aligned}$$



- Green-Lagrange Strain

$$\mathbf{E} = \frac{1}{2} (\mathbf{F}^T \mathbf{F} - \mathbf{1}) = \begin{bmatrix} 0.389 & 0 \\ 0 & -0.255 \end{bmatrix}$$

Example - Lagrangian Strain cont.

- Almansi Strain

$$\mathbf{b} = \mathbf{F} \cdot \mathbf{F}^T = \begin{bmatrix} 0.49 & 0 \\ 0 & 1.78 \end{bmatrix}$$

$$\mathbf{e} = \frac{1}{2}(\mathbf{1} - \mathbf{b}^{-1}) = \begin{bmatrix} -0.52 & 0 \\ 0 & 0.22 \end{bmatrix}$$

- Engineering Strain

$$\nabla_0 \mathbf{u} = \mathbf{F} - \mathbf{1} = \begin{bmatrix} -1 & -0.7 \\ 1.33 & -1 \end{bmatrix}$$

$$\mathbf{v} = \frac{1}{2}(\nabla_0 \mathbf{u} + \nabla_0 \mathbf{u}^T) = \begin{bmatrix} -1 & 0.32 \\ 0.32 & -1 \end{bmatrix}$$

Which strain is consistent with actual deformation?

Example - Uniaxial Tension

- Uniaxial tension of incompressible material ($\lambda_1 = \lambda \geq 1$)

- From incompressibility

$$\lambda_1 \lambda_2 \lambda_3 = 1 \Rightarrow \lambda_2 = \lambda_3 = \lambda^{-1/2}$$

$$x_1 = \lambda_1 X_1$$

$$x_2 = \lambda_2 X_2$$

$$x_3 = \lambda_3 X_3$$

- Deformation gradient and deformation tensor

$$\mathbf{F} = \begin{bmatrix} \lambda & 0 & 0 \\ 0 & \lambda^{-1/2} & 0 \\ 0 & 0 & \lambda^{-1/2} \end{bmatrix} \quad \mathbf{C} = \begin{bmatrix} \lambda^2 & 0 & 0 \\ 0 & \lambda^{-1} & 0 \\ 0 & 0 & \lambda^{-1} \end{bmatrix}$$

- G-L Strain

$$\mathbf{E} = \frac{1}{2} \begin{bmatrix} \lambda^2 - 1 & 0 & 0 \\ 0 & \lambda^{-1} - 1 & 0 \\ 0 & 0 & \lambda^{-1} - 1 \end{bmatrix}$$

Example - Uniaxial Tension

- Almansi Strain ($\mathbf{b} = \mathbf{C}$)

$$\mathbf{b}^{-1} = \begin{bmatrix} \lambda^{-2} & 0 & 0 \\ 0 & \lambda & 0 \\ 0 & 0 & \lambda \end{bmatrix} \quad \mathbf{e} = \frac{1}{2} \begin{bmatrix} 1 - \lambda^{-2} & 0 & 0 \\ 0 & 1 - \lambda & 0 \\ 0 & 0 & 1 - \lambda \end{bmatrix}$$

- Engineering Strain

$$\mathbf{v} = \begin{bmatrix} \lambda - 1 & 0 & 0 \\ 0 & \lambda^{-1/2} - 1 & 0 \\ 0 & 0 & \lambda^{-1/2} - 1 \end{bmatrix}$$

- Difference

$$\mathbf{E}_{11} = \frac{1}{2}(\lambda^2 - 1) \quad \mathbf{e}_{11} = \frac{1}{2}(1 - \lambda^{-2}) \quad \varepsilon_{11} = \lambda - 1$$

